

PDEs - DAY 21

Fourth order PDEs.

$$u_{tt} = -\alpha^2 u_{xxxx}$$

Vibrating beam problem.

The beam offers resistance to bending $\rightarrow$ fourth order.

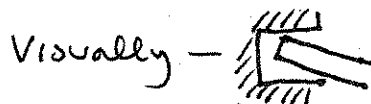
$$\alpha^2 = K/\rho = \frac{\text{rigidity}}{\text{linear density.}}$$

Assume the beam is simply fastened - held tight but the slope can change

$$\text{BC's} \rightarrow \left. \begin{array}{l} u(0,t) = 0 \\ u(1,t) = 0 \end{array} \right\} \text{held tight.}$$

problem... we need two more BC's.
we require that

$$\left. \begin{array}{l} u_{xx}(0,t) = 0 \\ u_{xx}(1,t) = 0 \end{array} \right\} \text{The curvature is zero - why. Zero bending Moment.}$$



when the beam moves the end will wiggle and try to remain straight.

$$\left. \begin{array}{l} u_t = -u_{xxxx} \\ u(0,t) = 0 \\ u(1,t) = 0 \\ u_{xx}(0,t) = 0 \\ u_{xx}(1,t) = 0 \\ u(x,0) = f(x) \\ u_t(x,0) = g(x) \end{array} \right\} \begin{array}{l} \text{let } \alpha = 1 \\ \text{Four BCs} \\ \text{Two ICs} \end{array}$$

Solve using separation of variables!

$$\text{let } u(x,t) = X(x)T(t) \text{ plug in}$$

$$\begin{aligned} T'X &= -X^{(4)}T \\ -\frac{T'}{T} &= \frac{X^{(4)}}{X} = K \end{aligned}$$

$$\left. \begin{array}{l} T'' + KT = 0 \\ X^{(4)} - KX = 0 \end{array} \right\} \begin{array}{l} \text{and} \\ \text{4 BC's.} \end{array} \text{ eigenvalue problem!}$$

NEW
PROBLEM

$$\left. \begin{aligned} X^{(4)} - kX &= 0 \\ X(0) &= 0 \\ X(1) &= 0 \\ X''(0) &= 0 \\ X''(1) &= 0 \end{aligned} \right\}$$

$$r^4 - k = 0$$

- we expect vibrating, sinusoidal
solns ... esp because of
the first two BC's.

let $k = \omega^2$ to make the calcs
easier ... assume $k \geq 0$

$$r^4 - \omega^2 = (r^2 + \omega)(r^2 - \omega) = 0$$

$$r^2 = -\omega$$

$$r^2 = \omega$$

or

$$r^2 = \pm i\sqrt{\omega}$$

$$r = \pm \sqrt{\omega}$$

~~not with~~

$\omega = 0$ Four repeated roots: $X(x) = Ax^3 + Bx^2 + Cx + D$

Easier to
apply conditions
at zero first

$$X(0) = D = 0$$

$$X''(0) = 2B = 0$$

$$X(x) = Ax^3 + Cx$$

$$X(1) = A + C = 0 \text{ or } A = -C \quad X(x) = C(x - x^3)$$

$$X''(1) = -6C = 0 \quad C = 0$$

TRIVIAL!

$\omega \neq 0$ then:

$$X(x) = A \sin(\sqrt{\omega}x) + B \cos(\sqrt{\omega}x) + \underbrace{C e^{\sqrt{\omega}x} + D e^{-\sqrt{\omega}x}}$$

we can also write
this as

$$C \sinh(\sqrt{\omega}x) + D \cosh(\sqrt{\omega}x).$$

Apply BC's.

$$X(0) = B + D = 0$$

$$X''(x) = -A\omega \sin(\sqrt{\omega}x) - B\omega \cos(\sqrt{\omega}x) + C\omega \sinh(\sqrt{\omega}x) + \omega D \cosh(\sqrt{\omega}x)$$

$$X''(0) = -B + D = 0$$

This means that $B = D = 0$!

$$X(x) = A \sin(\sqrt{\omega}x) + C \sinh(\sqrt{\omega}x).$$

$$X(1) = A \sin(\sqrt{\omega}) + C \sinh(\sqrt{\omega}) = 0$$

$$X''(1) = -A \sin(\sqrt{\omega}) + C \sinh(\sqrt{\omega}) = 0$$

$$A \sin(\sqrt{\omega}) = 0 \quad \text{and} \quad C \sinh(\sqrt{\omega}) = 0$$

$$\sqrt{\omega} = n\pi$$

$$C = 0$$

$$\text{or } \omega = (n\pi)^2$$

$$\text{or } k = (n\pi)^4$$

Notes on $\sinh(x)$ and $\cosh(x)$...

$$f(x) = \sinh(x) = \frac{1}{2}(e^x - e^{-x})$$

$$f(0) = 0$$

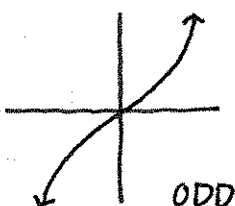
$$f'(x) = \cosh(x)$$

no negatives
out of
derivatives.

$$g(x) = \cosh(x) = \frac{1}{2}(e^x + e^{-x})$$

$$g(0) = 1$$

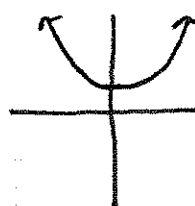
$$g'(x) = \sinh(x)$$



ODD

$$\sinh(x) = \text{sh}(x)$$

NOT PERIODIC



EVEN

$$\cosh(x) = \text{ch}(x).$$

Soln in X : $X(x) = A \sin(n\pi x)$ and $K = (n\pi)^4$

NOW SOLVE: $T'' + (n\pi)^4 T = 0$

$$T(t) = A \sin((n\pi)^2 t) + B \cos((n\pi)^2 t)$$

And $u(x, t) = \sum_{n=1}^{\infty} (A_n \sin[n^2 \pi^2 t] + B_n \cos[n^2 \pi^2 t]) \sin(n\pi x)$

notice the higher frequency
of vibration here!!

Now use ICs to ~~fix~~ solve for A_n and B_n .

$$\left. \begin{aligned} u(x, 0) &= \sum_{n=1}^{\infty} B_n \sin(n\pi x) = f(x) \\ u_t(x, 0) &= \sum_{n=1}^{\infty} n^2 \pi^2 A_n \sin(n\pi x) = g(x) \end{aligned} \right\} \text{we have solved this a million times.}$$

$$B_n = 2 \int_0^1 f(x) \sin(n\pi x) dx$$

$$A_n = \frac{2}{n^2 \pi^2} \int_0^1 g(x) \sin(n\pi x) dx$$

EX What if $u(x,0) = \sin(\pi x) + \frac{1}{2}\sin(3\pi x)$
 $u_t(x,0) = 0$

By observation $B_1 = 1$ $B_2 = 0$ $B_3 = \frac{1}{2}$ $B_4 = 0 \dots$
 $A_n = 0$

so our soln is...

$$u(x,t) = \cos(\pi^2 t) \sin(\pi x) + \frac{1}{2} \cos(9\pi^2 t) \sin(3\pi x)$$

{ compare This to the vibrating string
from last time...

$$u(x,t) = \cos(\pi t) \sin(\pi x) + \frac{1}{2} \cos(3\pi t) \sin(3\pi x)$$

MATLAB.

Think about this physically ... it makes sense.

Try another one... Left end rigidly fastened $u(0,t) = 0$
 $u_x(0,t) = 0$ } #4 from the homework.
Right end simply fastened $u(1,t) = 0$
 $u_{xx}(1,t) = 0$

Now we are solving:

$$\left. \begin{array}{l} X^{(4)} - kX = 0 \\ X(0) = 0 \\ X'(0) = 0 \\ X(1) = 0 \\ X''(1) = 0 \end{array} \right\} \begin{array}{l} \text{From the last example} \\ k = \omega^2 \\ X(x) = A \sin(\sqrt{\omega} x) + B \cos(\sqrt{\omega} x) + C \sinh(\sqrt{\omega} x) + D \cosh(\sqrt{\omega} x) \\ \text{OR} \\ X(x) = Ax^3 + Bx^2 + Cx + D. \end{array}$$

$\omega = 0$

$$\left. \begin{array}{l} X(0) = D = 0 \\ X'(0) = C = 0 \\ X(1) = A + B = 0 \quad A = -B \\ X''(1) = 6B = 0 \quad B = 0 \end{array} \right\} \begin{array}{l} X(x) = Ax^3 + Bx^2 \\ X(x) = B(x^2 - x^3) \end{array} \quad \text{still trivial.}$$

$\omega \neq 0$

$$\left. \begin{array}{l} X(0) = B + D = 0 \quad B = -D \\ X'(0) = A\sqrt{\omega} + C\sqrt{\omega} = 0 \quad A = -C \end{array} \right\} \begin{array}{l} X(x) = A(\sin(\sqrt{\omega} x) - \sinh(\sqrt{\omega} x)) \\ \quad + B(\cos(\sqrt{\omega} x) - \cosh(\sqrt{\omega} x)) \end{array}$$

$$X(1) = A(\sin(\tau\omega) - \sinh(\tau\omega)) + B(\cos(\tau\omega) - \cosh(\tau\omega)) = 0$$

$$X''(1) = A\omega(\sin(\tau\omega) + \sinh(\tau\omega)) - \omega B(\cos(\tau\omega) + \cosh(\tau\omega)) = 0$$

$$AS + BC = 0$$

$$-As^* - Bc^* = 0$$

$$A = -\frac{BC}{S}$$

$$\frac{BC}{S} s^* - Bc^* = 0$$

$$B\left(\frac{Cs^*}{S} - c^*\right) = 0$$

$B=0$ would
mean trivial.

$$\frac{Cs^*}{S} - c^* = 0 \Rightarrow \frac{(\cos(\tau\omega) - \cosh(\tau\omega))}{(\sin(\tau\omega) - \sinh(\tau\omega))} (\sin(\tau\omega) + \sinh(\tau\omega)) - (\cos(\tau\omega) + \cosh(\tau\omega)) = 0$$

$$\frac{\cos \tau\omega - \cosh \tau\omega}{\cos \tau\omega + \cosh \tau\omega} \cdot \frac{\sin \tau\omega + \sinh \tau\omega}{\sin \tau\omega - \sinh \tau\omega} = 1$$

$$\frac{\cos \tau\omega - \cosh \tau\omega}{\cos \tau\omega + \cosh \tau\omega} = \frac{\sin \tau\omega - \sinh \tau\omega}{\sin \tau\omega + \sinh \tau\omega}$$

$$\cancel{\cos \tau\omega} - \cosh \tau\omega = \cancel{\sin \tau\omega} - s$$

$$(\cos \tau\omega - \cosh \tau\omega)(\sin \tau\omega + \sinh \tau\omega) = (\sin \tau\omega - \sinh \tau\omega)(\cos \tau\omega + \cosh \tau\omega)$$

$$\begin{aligned} & \cancel{\cos \tau\omega} \sin \tau\omega - \cosh \tau\omega \sin \tau\omega + \sinh \tau\omega \cos \tau\omega - \cancel{\cosh \tau\omega} \sinh \tau\omega \\ = & \cancel{\sin \tau\omega} \cos \tau\omega - \sinh \tau\omega \cos \tau\omega + \sin \tau\omega \cosh \tau\omega - \cancel{\sinh \tau\omega} \cosh \tau\omega \end{aligned}$$

$$2 \cosh \tau\omega \sin \tau\omega - 2 \sinh \tau\omega \cos \tau\omega = 0$$

$$\cosh \tau\omega \sin \tau\omega = \sinh \tau\omega \cos \tau\omega$$

natural
frequencies
satisfy this.

The Lagrangian:

$$\mathcal{L} := \frac{1}{2} \mu \left(\frac{\partial w}{\partial t} \right)^2 - \frac{1}{2} EI \left(\frac{\partial^2 w}{\partial x^2} \right)^2$$

Assume no
External
Forces.

↑
Kinetic
Energy

↑
Potential
Energy
from Internal
forces

relationship between
curvature and
bending moment.

$$= \frac{1}{2} \mu \dot{w}^2 - \frac{EI}{2} w_{xx}^2$$

Euler-Lagrange:

$$\frac{\partial \mathcal{L}}{\partial w} - \frac{\partial}{\partial t} \left(\frac{\partial \mathcal{L}}{\partial \dot{w}} \right) + \frac{\partial^2}{\partial x^2} \left(\frac{\partial \mathcal{L}}{\partial w_{xx}} \right) = 0$$

$\downarrow$ $\downarrow$ $\downarrow$
 0 $\mu \dot{w}$ $-EI w_{xx}$

$$- \frac{\partial}{\partial t} \mu \dot{w} - \frac{\partial^2}{\partial x^2} (+EI w_{xx}) = 0$$

$$- \mu \ddot{w} - EI w_{xxxx} = 0$$

assuming a
homogeneous
beam.

$$\underline{\text{So}} \quad w_{tt} = -\alpha^2 w_{xxxx}$$

$$\text{where } \alpha^2 = \frac{EI}{\mu}$$

So the fourth derivative arises because the potential energy in the beam is dependent on curvature.

NOTE - Euler Lagrange eqn seeks to find the minimum path from an energy standpoint. In general.

$$\frac{\partial \mathcal{L}}{\partial f} - \frac{d}{dx} \frac{\partial \mathcal{L}}{\partial f'} + \frac{d^2}{dx^2} \left(\frac{\partial \mathcal{L}}{\partial f''} \right) - \dots + (-1)^n \frac{d^n}{dx^n} \left(\frac{\partial \mathcal{L}}{\partial f^{(n)}} \right) = 0$$

$$\text{where } \mathcal{L}(x, f, f', f'', \dots, f^{(n)})$$

in the most simple case → this can reduce to the statement that a max or min happens when the derivative is zero.